PATENT SPECIFICATION

(11) 1 264 568

DRAWINGS ATTACHED

- (21) Application No. 25014/70 (22) Filed 22 May 1970
- (45) Complete Specification published 23 Feb. 1972
- (51) International Classification B 29 c 27/06 B 65 b 51/14

(52) Index at acceptance

B8C 10A 10B1B 10B1D1 10D3B2 10K2 10T5 10U 3B1 3E2B B5K

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(54) HEAT SEALING APPARATUS

I, WILLIAM NELSON HISTED, a Canadian citizen, of 89 Empress Avenue, Hamilton, Ontario, Canada, do hereby declare the invention, for which I pray that a patent may be granted to me, and the method by which it is to be performed, to be particularly described in and by the following statement:

This invention relates to apparatus useful 10 for heat sealing films of the heat sealable type, such as polyethylene and polypropylene. This invention also relates to apparatus useful for the heat sealing and cutting films of a heat sealable material. In addition, this invention relates to apparatus useful for individually packaging articles in containers made of heat sealable films.

It is known that a pair of films made of a material such as polyethylene or poly-propylene may be sealed together effectively by the application of suitable amounts of heat and pressure. Heat sealable films of this type have desirable properties such as relatively high strength (as compared with paper of the same thickness), transparency and impermeability to water. Consequently such heat sealable films have been employed for the manufacture of bags and similar con-

Commonly used apparatus for the manufacture of bags or containers of heat sealable films includes a conveyor system for conveying a pair of heat sealable films positioned one on top of another to a heat sealing and cutting station. A reciprocating sealing and cutting blade is provided at the station and is heated to a suitable tempera-The conveyor system conveys a predetermined amount of the films, which usually have been previously sealed or otherwise joined together at one side, beyond the sealing and cutting station. This predeter-mined amount of the films is sufficient to construct one bag. When this predetermined amount of the films is in position, the conveyor system and hence the films stop. The sealing and cutting blade then descends upon

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and across the upper film while it is at rest. and pressure is applied to the films between the conveyor system and the blade. The result of this pressure and the heat of the blade is to seal the upper and lower films together. If sufficient heat and pressure are employed, the result is to cut the films as well as to seal the films on either side of the cut. blade then reciprocates upwardly and assumes its original position. As soon as the blade is withdrawn from contact with the film, the conveyor system moves forward again placing another predetermined amount of the films beyond the cutting and sealing station, and the operation is repeated. The completed bags are withdrawn from the apparatus by a conveyor.

A serious disadvantage of such an apparatus is the time consumed in making each bag. A large proportion of the time con-sumed is that which occurs as a result of the stoppage of the conveyor, the apparatus operating in regular intervals rather than

continuously.

In addition, heat sealable films such as a polyethylene and polypropylene can be printed. Bags or containers constructed of such films commonly carry printed advertis-ing matter or other printed matter, such as instructions for the use of an article which may be packaged therein. It has been common practice to print such matter on a continuous roll of the film, the printed matter being duplicated on each section of the continuous film from which a bag is to be made. The printed roll then has been conveyed to a bag-making machine such as the one previously mentioned. Because of the great disparity between the speed of operation of the printing press, which is relatively fast, and the relatively lower speed of operation of the bag-making machine, it has not been possible to lead the film directly from the printing press to the bag-making machine.

In Canadian Patent No. 651,984 issued November 13, 1962, to William N. Histed and entitled "Heat Sealing and Cutting Ap-



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paratus" there is described and claimed a heat sealing apparatus which overcomes the problems attendant upon the prior art apparatus by operating in a continuous manner. One embodiment of this apparatus consists of a supporting means adapted to support upper and lower heat scalable films at a scaling position with the upper film lying on top of the lower film, and heat sealing means 10 above the supporting means at the sealing position. The seat sealing means consists of a pivotably mounted sealing blade that is rotatable about an axis, means for completely rotating the sealing blade about such axis, and means for heating the sealing blade to a heat sealing temperature. The heat sealing blade is positioned such that during a portion of a rotation it is adapted to lie across and firmly contact the upper heat sealable film. The temperature of the heat sealing blade and the pressure between it and the supporting means is sufficient to heat seal the upper and lower films together.

While this apparatus has been found to be 25 a substantial improvement over the prior art apparatus of the type hereinbefore mentioned and to work satisfactorily in most operations, it does suffer from the defect that the height of articles which may be packaged using the apparatus is limited due to the circular movement of the sealing blade about its axis. In addition, there is a degree of "lost time" between one sealing operation and the next due to this circular

movement.

It is an object of the present invention to provide heat sealing apparatus which will accommodate larger articles than those that can be accommodated by the apparatus described in the above mentioned Canadian Patent No. 651,984.

Other objects of the invention will become apparent from the description which follows with reference to the accompanying drawings, in which:

Figure 1 is a perspective view of apparatus embodying the present invention;

Figure 2 is a side elevational view of the apparatus of Figure 1 with parts removed

50 for clarity; Figure 3 is a close-up perspective view of

apparatus that may be used in practising the present invention;

Figure 4 is a sectional view of a heat seal-55 ing and cutting blade taken along line 4-4 of Figure 1:

Figure 5 is a sectional view of a heat sealing and cutting blade taken along line 5-5 of Figure 3;

Figure 6 is an alternative embodiment, in section, of the heat sealing and cutting blade shown in Figure 5; and

Figure 7 is another alternative embodiment, in section, of the heat sealing and cut-65 ting blade shown in Figure 5.

In the drawings, referring particularly to Figures 1, 2 and 4, a heat sealing and cutting apparatus 10 consists of a roller 12 rotatably mounted between two longitudinal members 13 (only one of which is shown) of a frame 106. The roller 12 is mounted between the ends of two longitudinally aligned conveyor belts 14 and 16 having driven rollers 18 and 20 respectively, the drives for these rollers not being shown. The roller 12 is mounted on each longitudinal member 13 by means of a bearing 22 and the shaft 24 of roller 12 is journalled in bearings 22. A series of freely rotatable rollers 28 consituting a transfer table are mounted between the end of conveyor 14 and roller 12 and a freely rotatable roller 30 is mounted between roller 12 and conveyor 16. Rollers 18, 28, 12, 30 and 20 are arranged to have a common tangent.

A heat sealing and cutting member 32 consists of an elongated inverted channel member 34 and an elongated blade supporting member 36. The support member 36 is fixed at both ends to upwardly projecting arm members or straddle bars 38, 40. A blade member 42 is fixed to the support member 36 and is embedded in electrical insulating material 37. Rods 44, 46 attached to blade supporting member 36 pass upwardly through the channel member 34 and terminate in heads 48, 50 respectively of larger diameter than the rods. Positioned between heads 48 and 50 and the upper surface of the channel member 34 are compression 100 springs 52 and 54 respectively shown in their normally uncompressed position in Figure Welded or otherwise attached to the channel member 34 adjacent each rod 44, 46 are a pair of downwardly projecting arms 105 56, 58 which terminate at their lower ends in two elongated tube members 60, 62 which are parallel to the blade member 42. Tube members 60 and 62 have water or other cooling liquid flowing therethrough when the 110 apparatus is in operation, and this cooling liquid is fed through pipes 64, 66 and 68, 70 respectively. The blade member 42 is heated electrically and is connected to a suitable power supply (not shown) by conductors 72, 115 only one of which is shown in Figure 4.

The instant invention is directed to the apparatus for moving the heat sealing and cutting apparatus 32, and this now will be described. The arms 38, 40 each are constructed 120 with an clongated slot 74, 76 respectively through which shaft 24 extends. At their upper ends each of arms 38, 40 is pivotally connected to lever arms 78, 80 by pivot pins 32, 84 respectively. Each of lever arms 78, 125 80 is pivotally connected to a link 36, 88 by means of pivot pins 90, 92 respectively. In turn, links 86, 88 are pivotally connected to adjustable bracket members 94, 96 by pivot pins 98, 100 respectively. The adjust- 130 able bracket members 94, 96 are bolted to longitudinal members 102, 104 of a frame 106 by bolts 105, 107 that pass through slots 109, 111 formed in the bracket members 94,

96 respectively. At a point intermediate pivot pins 82 and 90, lever arm 78 is connected by means of a pivot pin 108 to a crank wheel 110. Lever arm 80 is similarly connected by means of a pivot pin 112 to a crank wheel 114. Each of crank wheels 110, 114 is provided with a counterweight 116 (only one of which is shown in Figures 1 and 2) opposite pivot pins 108 and 112 respectively. Crank wheels 110, 114 are fixed on each end of a shaft 118 which is journalled adjacent crank wheel 114 in a bearing 120. Shaft 118 is similarly journalled adjacent crank wheel 110 in a bearing (not shown). Bearing 120 is fixed to a further longitudinal member 122 of frame 106. The other bearing is fixed to a longitudinal member 124 of frame 106. Constituting pivot pins 82 and 84 and extending between straddle bars 38, 40 is a cranked rod 126. Fixed to shaft 118 adjacent crank wheel 110 are a pair of cog wheels 128, 130 shown in dotted outline in Figure 1. Cog wheel 128 is connected by a chain 132 to a further cog wheel 134 fixed to cranked rod 126 adjacent lever arm 78. Cog wheel 130 is connected by a chain 136 to a cog wheel 138 fixed on the drive shaft 140 of a gear reduction unit (not shown) of an electric motor (not shown). Chain 136 also passes around an idler cog 35 wheel 146 secured on a shaft 148, the shaft 148 being rotatably mounted in a bearing (not shown) fixed to an upright member (not shown) of the frame 106. Chain 136 also engages a gear wheel 150 integral with the roller 12 and passes over an idler cog wheel 142. Chain 136 also may pass over a cog wheel connected to roller 18 to drive this roller. The shaft 148 also may have mounted thereon pulleys or cog wheels whereby belts or chains may drive reels of heat scalable film such as reel 152 shown schematically in Figure 2.

Turning now to the modification of Figures 3 and 5, like reference numerals have been used to indicate the same elements as are illustrated in Figures 1, 2 and 4. The modification illustrated in these Figures concerns primarily the construction of the heat sealing and cutting member 32. The cranked rod 126 has been omitted from the apparatus of Figure 3 for the sake of simplicity. In the modification of Figures 3 and 5 the heat scaling and cutting member consists of an inverted channel member 154 (partially cut away for clarity) and an elongated blade supporting member 156 fixed to the straddle bars 38, 40 and to which is attached an elongated electrical insulator 158. Embedded in the insulator 158 is a heat sealing and cutting blade 160. A pair of rods 161, 162

pass upwardly from the blade supporting member 156 through the channel member 154 and terminate in heads 164, 166 of larger diameter than rods 161, 162. Positioned between the channel member 154 and the heads 164, 166 are compression springs 168, 170 respectively. A pivot pin 172 is fixed to rod 161 between the underside of the channel member 154 and the blade supporting member 156. Pivotally mounted on pivot pin 172 are links 174, 176. To the other extremities of links 174, 176 are pivoted downwardly projecting arms 178, 180 by means of pivot pins 182, 184 respectively. Each of arms 178, 180 is provided with an elongated slot, only one of which, 136, formed in arm 178 is shown in Figure 3. An identical slot is formed in arm 180. Headed pins 188, 190 are secured to insulator 158 and pass through the slots formed in the arms 178, 180 respectively. A similar arrangement of arms, pivot pins, links and headed pins is associated with rod 162, only partially shown in Figure 3 as link 192, pivot pin 193, arms 194, 196, slot 198 and headed pin 200. Arms 178, 194 terminate at their lower ends in an elongated tube member 202. Arms 180, 196 terminate at their lower ends in an elongated tube member 204. Tube members 202, 204 are parallel to the blade member 160 and have water or other cooling liquid flowing therethrough when the apparatus is in operation. This cooling liquid is fed to tube members 202, 204 by pipes 206, 203 and 210, 212 respectively. The blade 100 member 160 is heated electrically and is connected to a suitable power supply by conductors 214 (only one shown).

In the embodiment of Figure 6, there is shown a modification of the lower end of 105 the heat sealing and cutting member shown in detail in Figure 5. Like reference numerals to describe like elements to those of Figure 5 are used where appropriate. Arms 178, 180 terminate in solid rods 216, 218 110 which extend parallel to the cutting and sealing blade 160 in like manner to pipes 202, 204 in Figure 5. Located adjacent the blade 160 are pipes 220, 222 which run parallel to the blade 160 throughout its length and 115 parallel to rods 216, 218. Pipes 220, 222 are adapted to carry cooling liquid such as water. The cooling liquid is fed to the pipes 220, 222 in the same manner as cooling liquid is

fed to pipes 202, 204 in Figure 3.

In Figure 7 a further modification of the structure shown in Figure 5 is illustrated. Like reference numerals to describe like elements to those of Figure 5 are used where appropriate. Arms 173, 180 terminate in 125 coolant tubes 224, 226 which extend parallel to the cutting and sealing blade 160 in like manner to the pipes 202, 204 in Figure 3. Located on either side of the blade 160 parallel thereto and co-extensive therewith 130

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are two heat sealing blades 228, 230. The blades 228, 230 are slightly shorter than blade 160 and may be corrugated in configuration to give a wider and better seal than a straight line seal. Blade 160 is directly electrically heated by means not shown, whereas the blades 228, 230 are heated indirectly by the blade 160. The effect of this indirect heating is that blades 228, 239 only seal films with which they come into contact and do not cut them, whereby the width of

the seal may be increased. In operation, firstly with reference to the apparatus illustrated in Figures 1, 2 and 4, an item to be packaged, such as that indicated as 232 and appearing in chain outline, is fed by conveyor 14 onto rollers 28. Also fed onto rollers 28 from beneath and through the gap between the conveyor 14 and rollers 28 is a film of heat sealable material 234 (see Figure 2) not shown in Figure 1 but indicated by a broad arrow. The item 232 thus sits on the film. A further film of heat sealable material 236 (not shown in 25 Figure 1 but indicated by a further broad arrow) is fed from reel 152 (not shown in Figure 1) towards the heat sealing and cut-ting member 32. With the apparatus in the position indicated in Figures 1 and 2 scaling together and cutting of the films occurs. Thus, assuming that one article to be packaged is beyond the heat sealing and cutting member 32, say on conveyor 16, and a second article is located on rollers 28, the films 234 and 236 are held in contact with each other by pipes 60, 62, which cool the film in that area by the passage of cooling liquid therethrough. Heated blade 42 melts the films in the area beneath the blade and cuts through them. Since this operation is not instantaneous, provision must be made to hold the cutting and sealing blade 42 in contact with the films for a short time while the roller 12 is continuously rotating in response to movement of drive shaft 140 transmitted to roller 12 through cogs 150, 142 and chain 136. This dwell time is provided by pivotal movement of links 86, 83 about pivot pins 98, 100 (see Figure 2). As crank wheels 110, 114 rotate in the direction of the arrows in response to movement of drive shaft 140, lever arms 78, 80 pivoting about pins 108, 112 cause pivotal movement of arms 38, 40 and thus of the heat sealing and cutting member 32 secured thereto about pivot pins 82, 84, Links 86, 88 pivot about pivot pins 98, 100 during movement of lever arms 78, 80. This movement occurs between the position shown in solid line in Figure 2 and the position shown in dotted line generally indicated as 238 in Figure 2. Since the support member of the heating and cutting member 32 is fixed to the straddle bars 38, 40, the blade 42 is held in contact with the films as the roller 12 moves. The time during which the blade 42 is held in contact with the films is predetermined and is sufficient to effectively seal the films together and to cut the films between the scal. It may be desired simply to seal the films: 70 rather than to cut and seal them.

This may be achieved by adjusting the temperature of the blade or the pressure exerted on the films between the blade and roller 12. Adjustment of the apparatus to achieve different pressures will be described

While the heat sealing and cutting member 32 is in its sealing and cutting position as shown in Figures 1 and 2, the tubes 60, 62 and arms 56, 58 are forced upwardly by the roller 12, thereby compressing springs 52, 54 between inverted channel 34 and heads 48, 50. In the normal uncompressed position of compression springs 52, 54 tubes 60, 62 project below the level at which they would be were roller 12 contacting them and This is below the level of the blade 42. illustrated by the chain outline of the roller 12 in Figure 4. When the springs are compressed, pipes 60, 62 and the channel 34 assume the position shown in dotted outline in Figure 4. The pipes then hold the films firmly in contact with the roller 12 during the sealing and cutting operation and relieve any tension in the part of the films being sealed and cut that might otherwise occur by movement of the previously sealed object away from the heat sealing and cutting station. Projection of the pipes 60, 62 below 100 the level of the blade 42 acts to prevent premature contact between the blade and upper film 236 as film 236 passes beneath the blade prior to assumption of the desired sealing position in contact with roller 105

As the crank wheels 110, 114 continue to rotate, the lever arms 78, 80 are raised, as are straddle bars 38, 40. Eventually pivot pins 82, 84 reach a point of maximum eleva- 110 tion above the roller 12 and then descend to the original position shown in full line in Figure 2. As the lever arms 78, 80 are raised and lowered and the straddle bars 38, 40 are raised and lowered, links 86, 88 pivot 115 backwards and forwards about pivot pins 98, 100, pivot pins 90, 92 describing the arc of a circle during such movement. pivot pins 82. 84 follow substantially the path of an ellipse, as is indicated by the 120 chain line 240. The extremities of the major axis of the ellipse are the points of closest and farthest approach of the pivot pins 82, 84 to the roller 12. The position of the arm 80, and straddle bar 40 in an intermediate 125 position are shown fragmentarily in the broken line 242. As the straddle bars 38, 40 rise, heat sealing and cutting member 32 rises allowing the films together with the material packaged therebetween to be con- 130

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veyed by roller 12 onto conveyor 16. The heat sealing and cutting member 32 returns to its sealing and cutting position when a predetermined interval in which the article to be packaged has passed has elapsed and the films required to be sealed and cut have been conveyed into position. It will be seen that because of the elliptical path followed by the pivot pins 82, 84, bulky objects may be packaged between the films without the sealing and cutting member 32 accidentally engaging the object to be pack-This is in contrast to the apparatus described in the aforementioned Canadian Patent No. 651,984 where, due to circular movement of the sealing and cutting blade, the height of articles which may be packaged in a tight package is limited. The shape of the ellipse may be varied in the manner hereinafter described.

During operation cog wheel 128 rotates, and cranked rod 126 is rotated via cog 134 and chain 132. The cranked rod 126 serves to prevent the film 236 from engaging the side of the sealing and cutting element 32 during downward movement of the element 32. The film is engaged by the cranked rod 126 and by the tube 60. The operation of cranked rod 126 is illustrated in the broken line outline 242. This cranked rod may be omitted, such as in the apparatus illustrated in Figure 3, depending on the angle at which the film 236 approaches the roller 12. the angle is so steep that the side of member 32 would foul the film 236 then rod 126 should be employed. If the angle is not so steep, then the rod may be omitted. Cranked rod 126 has an additional use. It is well known that heat sealable films such as polyethylene and polypropylene readily pick up static electricity which may be undesirable. The cranked rod 126 may be coated with an antistatic material so that the static is removed from the film 236 by wiping of the rod 126 against the film, and, to this end, rod 126 may be grounded.

Turning now to the operation of the apparatus of Figures 3 and 5, this operates in identical manner to the apparatus of Figure 1 with the exception of the operation of the heat sealing and cutting member 32, and, therefore, only the action of the sealing and cutting member will be described. As has already been noted, the apparatus illustrated in Figure 3, does not include the cranked rod 126. This may, of course, be included in the embodiment of Figure 3 if desired either to prevent fouling of the sealing and cutting member 32 by the film 236 or to remove static electricity from the film, or both.

The heat sealing and cutting member 32 is shown in a compressed position in Figure 5 with compression spring 168 in compression between the head 164 and the channel member 154. Spring 170 is similarly in compres-

sion between head 166 and the channel member 154. The compression of spring 168 is caused by the upward pressure of the roller 12 on the arms 178, 180. In the relaxed position arms 178, 180 hang substantially parallel to each other and extend downwardly below the level of the blade 160 in similar manner to arms 56 and 58 in Figure The upward pressure on the arms causes the links 174, 176 to pivot about the pivot 172 while arms 178, 180 pivot about pivot pins 182, 184. Links 176, 174 bear against the downwardly projecting sides of the inverted channel member 154, thereby causing compression of spring 168. As a result of the foregoing movements, tubes 202, 204 move towards each other. Movement of the tubes 202, 204 towards each other has the effect of releasing tension present in the films being sealed together. This gives a more efficient seal and allows the films to be readily cut. It will be understood that the links and arms associated with rod 162 operate in the same way as the links and arms associated with rod 161.

As in the case of the apparatus illustrated in Figure 4, when the tubes 202, 204 no longer engage the films in contact with the roller 12, springs 168, 170 return the arms to their normal position parallel to each other, and the tubes 202, 204 serve to prevent premature contact between the film 236 and the blade 160.

The heat sealing and cutting member illustrated in Figure 6 has an identical operation 100 to that of the member shown in Figure 5 with the exception that it is the solid rods 216, 218 attached to the lower end of arms 178, 180 which move towards each other releasing tension in the films. The cooling 105 pipes 220, 222 serve only to cool the film adjacent the blade 160 so that a so-called rotten seal" is avoided.

The heat sealing and cutting member illustrated in Figure 7 has an identical oper- 110 ation to that of the member shown in Figure 6 with coolant tubes 224, 226 moving towards each other to relieve tension in the films. The use of three blades 160, 228 and 230 has the effect of providing a wider seal. 115 As indicated above, only blade 160 is directly heated, and this blade both seals and cuts the films, whereas blades 228 and 230 are heated indirectly by blade 160 and only seal the films but do not cut them. Of course, if desired, blades 228 and 230 also can be connected to a series of electrical energy and heated directly rather than by heat transfer from blade 160.

As in the case of Figure 5, when the rods 125 216, 218 in Figure 6 and tubes 224, 226 in Figure 7 no longer engage the films in contact with the roller 12, arms 178, 180 assume a position parallel to each other. The rods then serve to prevent premature 130

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contact between the film 236 and the blade

Various adjustments may be made to the apparatus to suit individual requirements as will now be discussed.

The pressure exerted by the sealing and cutting element 32 on the films on roller 12 at the sealing and cutting station may be adjusted by adjusting the position of bracket members 94, 96 on frame 106. In the arrangement illustrated in Figure 1, if the bracket members 94, 96 are raised, the pressure exerted by the sealing and cutting member 32 will increase, and if bracket members 94, 96 are lowered, the pressure exerted will decrease. In this way, the pressure on the films at the cutting and sealing station can be adjusted to achieve the desired require-

Movement of pivot pins 108, 112 radially inwardly on crank wheels 110, 114 respectively and movement of shaft 118 and hence the crank wheels towards sealing and cutting member 32 has the effect of decreasing the height of the path along which the sealing and cutting member 32 travels. Similarly, movement of the pins 108, 112 radially outwardly and movement of shaft 118 away from sealing and cutting member 32 increases the height of elliptical path 240. Both operations are necessary, i.e. movement of the pivot pins combined with movement of the shaft, to maintain uniform pressure between the sealing and cutting member 32 and roller 12 over the period taken for the sealing and cutting operation.

Modifications are possible to the apparatus illustrated in the drawings, as will be obvious

to those skilled in the art.

One such modification consists in removal of lever arms 78, 80 completely together with rod 126, movement of crank wheels 110, 114 to a position over roller 12, and connection of pins 108, 112 to straddle bars 38, 40, pivot pins 108, 112 taking the place of pins 82, This modification represents the extremity of the movement of shaft 118 mentioned in the previous paragraph. In order to maintain uniform pressure, the pins must be moved radially inwardly by a compensatory amount.

A further modification consists in providing a lug fixed to each of crank wheels 110, 114 and projecting radially outwardly therefrom. Pivot pins, such as pivot pins 108, 112 are located at the outer extremities of the lugs and rigid arms are provided extending between these pins and pins 82, 84, these arms provided on pins 82, 84 and the pins of the lugs. Lever arms 78, 80 are disconnected from crank wheel 110, 114 and the end of the lever arms adjacent straddle bars 38, 40 are provided with elongated slots running parallel to the length dimension of the 65 lever arms and through which pins 82, 84 extend. This modification has the effect of increasing the height of travel of the member 32, thereby increasing the height of articles which may be packaged. The same effect could be achieved by using larger crank wheels 110, 114. A compensatory movement of the shaft 118 away from the member 32 is necessary to maintain uniform pressure at the sealing station, as explained

The apparatus as described will only seal and cut the ends of the package. The sealing of the sides of the package may be achieved by methods known in the art, such as by the method described in the abovementioned Canadian Patent No. 651,984.

WHAT I CLAIM IS:—

1. A heat sealing apparatus comprising feed means adapted to continuously feed upper and lower heat sealable films to a sealing position, the upper film lying on top of the lower film at the sealing position, the films being supported on a surface at the sealing position, a heat sealer adapted to seal the films together at the sealing position, the heat sealer comprising at least one heat sealing blade mounted for limited pivotal movement about a first axis, means for moving the first axis along a substantially elliptical path the extremities of the major axis of which are constituted by the points of nearest and farthest approache of the axis of the sealing position, the movement being in a direction such that at the sealing position the blade and the films travel in substantially 100 the same direction, and heating means for heating the blade to a heat sealing temperature, the blade being adapted to lie across and firmly contact the upper film at the sealing position, the temperature of the blade 105 and the pressure of the blade on the films at the sealing position being sufficient to heat seat the upper and lower films together.

An apparatus as claimed in claim 1 in which the major axis is substantially per- 110 pendicular to the surface at the scaling posi-

3. A heat sealing apparatus comprising feed means adapted to continuously feed upper and lower heat sealable films to a sealing position, the upper film lying on top of the lower film at the sealing position; a heat sealer adapted to seal the films together at the sealing position, the heat sealer comprising at least one heat sealing blade; means 120 for moving the heat sealing blade, the move-ment of the heat sealing blade being in a direction such that at the sealing position the blade and the films travel in substantially the same direction, the means for moving the 125 heat sealing blade comprising a lever member having two ends, a crank member rotatable about an axis of rotation and pivotally connected to the lever member about a first

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pivot axis intermediate the ends of the lever member and offset from but parallel to the axis of rotation, an arm pivotally connected to the lever member about a second pivot axis on one side of and parallel to the first pivot axis, means cooperating with the arm at a point remote from the lever member and establishing a third pivot axis for the arm remote from but parallel to the second pivot axis, means connected to the lever member at a point on the opposite side of the first pivot axis to the one side and constraining the point to move back and forth in an arc of a circle during movement of the lever member, means for rotating the crank member, the blade being carried by the arm, and heating means for heating the blade to a heat sealing temperature, the blade being adapted to lie across and firmly contact the upper film at the sealing position, the tem-perature of the blade and the pressure of the blade on the films at the sealing position being sufficient to heat seal the upper and lower films together.

4. An apparatus as claimed in claim 3 in which the arm has an elongated slot therein, the means cooperating with the arm comprising a member extending through said slot and on which the arm slides during move-

0 ment of the arm.

5. An apparatus as claimed in claim 3, or 4, in which the means connected to the

lever arm at the point comprise a link pivotally connected to the lever arm at the point and also pivotally connected to a support member at a fourth pivot axis to oscillate back and forth about the fourth pivot axis during movement of the lever arm, the fourth pivot axis being parallel to the first pivot axis.

6. An apparatus as claimed in any of claims 1 to 5 including a rotatable roller located below the films and over which the

films pass at the sealing position.

7. An apparatus as claimed in claim 4 or 5 including a rotatable roller located below the films and over which the films pass at the sealing position and having a shaft, the shaft constituting the member extending through the slot.

8. The apparatus as claimed in claim 1 substantially as hereinbefore described with reference to and as illustrated in any of the

accompanying drawings.

9. The apparatus as claimed in claim 3 substantially as hereinbefore described with reference to and as illustrated in any of the accompanying drawings.

HASELTINE LAKE & CO., Chartered Patent Agents, 28 Southampton Buildings, Chancery Lane, London, W.C.2, Agents for the Applicant.

Printed for Her Majesty's Stationery Office by Burgess & Son (Abingdon), Ltd.—1972.

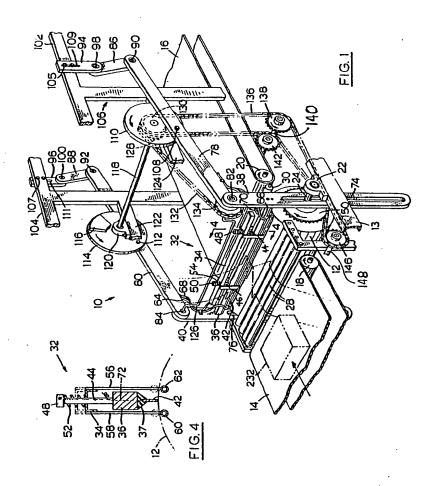
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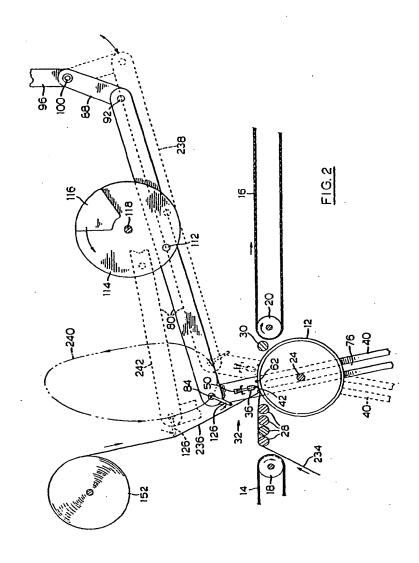
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